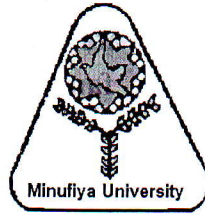


Minoufia University
Faculty of Engineering
Mechanical Power Eng. Dept
Academic Year: 2017-2018
Date: 30-12-2017



Subject: Industrial Ventilation
Code: MPE 502
Academic level: Diploma.
Time allowed: 3 hours
Total degree: 100 marks

Answer all the following questions:

Question-1

[25marks]

- a- Particulate matter one of the main sources of the indoor air pollution, classify solid particles contaminants and mentions its hazards on the human health.
- b- Explain with sketch the different Methods to control contaminants of air inside closed spaces.
- c- Describe using diagrammatic sketch the natural ventilation system and report its advantages and disadvantages.
- d- Show the difference between the three methods of industrial ventilation system (i.e. dilution and local exhaust), and mention only the disadvantages of each method.

Question-2

[25 marks]

- a- What are the chief requirements of an air ventilation duct systems?
- b- Discuss the difference between the fans, blower and the compressor.
- c- Explain with sketch the difference type of an axial fans and mention advantages and disadvantage of each type.
- d- Explain with the aid of a diagram the performance curve of a fan under specific conditions of fan (volume flow rate, system static pressure and efficiency).
- e- Discuss the purposes of hood ventilation and the meaning of capture velocity. Also, explain with sketch the different types of Ventilation hoods.

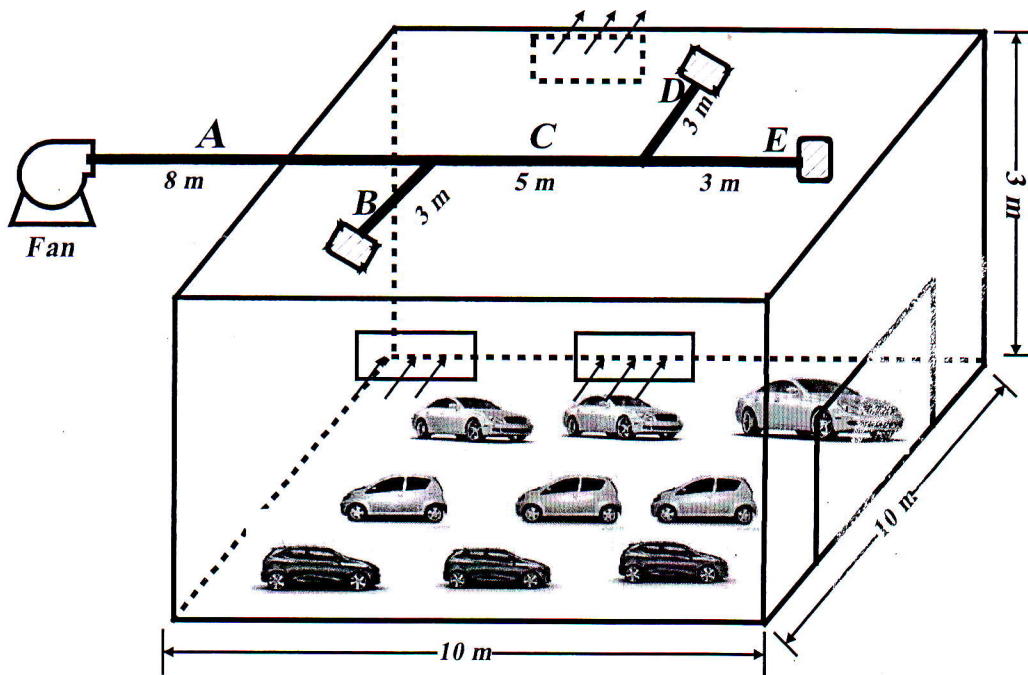
Question-3

[50 marks]

- A garage has the dimensions as shown in the next figure. The inlet and exit openings have the same shape and the same dimensions (40 cm×125 cm). The difference height between inlet and exit opening (i.e. H) is 1.2 m and the discharge coefficient C_D is 0.61 for all the openings. The dynamic pressure head at the inlet

opening is neglected. The average temperature inside the garage is 38 °C while the outside temperature is 27°C. The air flow rate from duct openings are $Q_B=30\%$ of the fan flow rate and $Q_D=Q_E=35\%$ of the fan flow rate. Take the pressure loss coefficients at bend is 0.8 and the exit is 1.0 along the duct. Also, take the velocity for the main duct is 8 m/s and the fan efficiency is 0.82. Assume any required data and calculate the following:

- i- The required air flow rate if ACR=6.
- ii- The natural flow rate based on the buoyancy effect only.
- iii- Design the ventilation duct by using the equal friction coefficient method.
- iv- The pressure losses at the exit damper.
- v- The fan horse Power (HP).



Use the following relations if you need:

$$\frac{\Delta P_f}{L} = \frac{0.022243 \dot{Q}_{air}^{1.85}}{D^{4.973}}, \quad D_{eq,B} = D_{eq,A} \left(\frac{Q_B}{Q_A} \right)^{\left(\frac{1.85}{4.973} \right)}, \quad D_{eq} = 1.3 \frac{(ab)^{0.625}}{(a+b)^{0.25}}$$

$$\dot{V} = A_e C_e \sqrt{\frac{2g\Delta H\Delta T}{T}}, \quad \frac{1}{(A_e C_e)^2} = \frac{1}{(\sum A_i C_i)^2} + \frac{1}{(\sum A_o C_o)^2}$$

With best wishes

Dr. Ashraf Amin